

A Modified Harmonic Balance Method for Solving Strongly Generalized Nonlinear Damped Forced Vibration Systems

M. Wali Ullah 1,2 , M. Alhaz Uddin 1,* and M. Saifur Rahman 3

Received: May 3, 2021; Revised: September 1, 2021

Abstract: A modified harmonic balance method is proposed for solving damped forced generalized nonlinear oscillators with strong nonlinearity. In the classical harmonic balance method, a set nonlinear algebraic equation of the unknown coefficients is solved by the numerical method to determine the unknown coefficients. However, in the present method, only one nonlinear equation and a set of linear algebraic equations are required for solution, thereby reducing the computational effort. Comparison between the results obtain by the proposed method and the numerical method is presented in figures which show a good agreement with the numerical results. The proposed method can play an important role for handling such nonlinear dynamical systems.

Keywords: harmonic balance method; generalized nonlinear damped oscillators; forcing term.

Mathematics Subject Classification (2010): 34E05, 34E10, 34M10.

 $^{^1}$ Department of Mathematics, Khulna University of Engineering & Technology, Khulna-9203, Bangladesh.

² Department of Computer Science & Engineering, Northern University of Business and Technology Khulna, Bangladesh.

³ Department of Mathematics, Rajshahi University of Engineering & Technology Rajshahi-6205, Bangladesh.

^{*} Corresponding author: mailto:alhazuddin@math.kuet.ac.bd

1 Introduction

Nonlinear oscillators are very important in many areas of applied mathematics, physics, and engineering. Most of the physical problems are governed by the nonlinear differential equations. The exact solutions of these nonlinear equations are rarely obtained. Therefore, many researchers focused on analytical approximation methods. Among them, the perturbation method [1,2], homotopy analysis method [3,4], homotopy perturbation method [5-7], variational iteration technique [8,9], harmonic balance method (HBM)[10-13] are well known. The perturbation methods [14–20] are widely used techniques for dealing with the nonlinear differential systems and they were originally developed for weakly nonlinear dynamical systems. Jones [20] modified the perturbation method to extend the accuracy of the solution when the parameter was not small. Further, a modification of the Lindstedt-Poincare technique was presented by Cheung et al. [21] based on the Jones technique [20. The modified Lindstedt-Poincare method has been generalized by Alam, et al. [22] and it is applicable for a wide variety of nonlinear oscillators. The harmonic balance method (HBM) is another powerful technique to obtain the periodic solution of the nonlinear oscillators. According to this method, the solution is chosen as a truncated Fourier series. Usually, a set of strongly nonlinear algebraic equations appears among the unknown coefficients of several harmonic terms and these equations are solved by the numerical method. Further, this method has been modified by several researchers [23–28]. Rahman et al. [23] used a modified HBM to study the Van der Pol equation. Rahman and Lee [26] developed a modified residue HBM to handle nonlinear vibrating problems of beam. Wu [27] developed the harmonic balance method for the Yao-Cheng oscillator. Wagner and Lentz [28] developed a HBM to handle the Duffing oscillator with a forcing term with cubic nonlinearity. Younesian et al. [29] applied He's frequencyamplitude formulation and He's energy balance method to handle strongly nonlinear the generalized Duffing oscillators without forcing term. Uddin et al.[30] presented an analytical approximation technique for handling the generalized nonlinear Duffing equation with strong nonlinearity without external forcing term. Rafieipour et al. [31] developed an analytical approximate solution for the generalized nonlinear vibration of a micro electro mechanical system by using He's frequency amplitude formulation. Karahan and Pakdemirli [32] studied free and forced vibration response of the strongly nonlinear cubic-quintic Duffing oscillators by using the multiple time scale method. Ullah et al. [33] developed a modified harmonic balance method to handle nonlinear oscillators with cubic nonlinearity in the presence of external forcing term. Rahman et al. [34] presented a modified harmonic balance method to solve the nonlinear vibration problem of a beam resting on nonlinear foundation. Recently, Yeasmin et al. [35] have presented an analytical technique for handing the quadratic nonlinear oscillator based on the harmonic balance method for free vibration nonlinear problems. Cheib et al. [36] presented an analysis of the dynamics of a two-degree-of-freedom nonlinear mechanical system under harmonic excitation. It is noticed that the approximate analytical techniques for solving the damped forced generalized nonlinear oscillators with strong nonlinearity are almost untouched. To fill this gap, a modified harmonic balance method has been presented for handling strongly generalized nonlinear damped forced oscillators. The convenience of this method is that only one nonlinear algebraic equation and a set of linear algebraic equations are required to solve by the numerical method, which reduces the heavy computational effort that is required in classical harmonic balance methods. The obtained results are compared with the corresponding numerical results in graphs and it shows a good agreement with these numerical results.

2 Method

Let us assume a strongly generalized nonlinear damped forced oscillator [29–33] of the form

$$\ddot{x} + \mu \dot{x} + \omega_0^2 x + \epsilon (\alpha_3 f_3(x) + \alpha_5 f_5(x) + \dots + \alpha_n f_n(x)) = E \cos(\omega t), \tag{1}$$

where over-dots denote differentiation with respect to time t, ω_0 is the natural frequency, μ is the linear damping coefficient, $f_i(x)$ are given general nonlinear functions of x, α_i (i=1,3,5,...n) are constants, ϵ is a positive parameter which is not necessarily small, E is the amplitude of the excitation force and ω is the forcing frequency. All of the parameters are positive. We assume that $\mu=0$ in our idealized systems. But damped motion is important for most of the physical and engineering vibration problems. In this paper, we are going to assume that $\mu\neq 0$. This is a non-autonomous system since time t explicitly appears in the right-hand side of the given equation. In particular, periodically forced harmonic oscillators depended explicitly on time t and exhibited quite interesting behavior. When a damped Duffing-type oscillator is driven with a periodic forcing function, the result may be a periodic response at the same frequency as the forcing function. Since the unforced oscillation is the dissipated energy due to the damping, we are not surprised to find that it is absent from the steady state forced behavior. According to the proposed method, the approximate solution of Eq.(1) is assumed [33] in the following form:

$$x = a\cos(\omega t) + b\sin(\omega t) + a_3\cos(3\omega t) + b_3\sin(3\omega t) + \dots,$$
 (2)

where $a, b, a_3,b_3...$ are the unknown coefficients. Now, differentiating Eq.(2) twice with respect to t and then putting into Eq.(1) and expanding $f_i(x)$ as a truncated Fourier series expansion and taking the coefficients of equal harmonics from both sides, we obtain the following set of algebraic equations:

$$a(-\omega^2 + \omega_0^2) + b\mu\omega + \epsilon C_1(a, b, a_3, b_3, ...) = E,$$
 (3a)

$$a(-\omega^2 + \omega_0^2) - a\mu\omega + \epsilon S_1(a, b, a_3, b_3, \dots) = 0,$$
(3b)

$$a_3(-9\omega^2 + \omega_0^2) + 3b_3\mu\omega + \epsilon C_3(a, b, a_3, b_3, \dots) = 0,$$
(3c)

$$b_3(-9\omega^2 + \omega_0^2) - 3a_3\mu\omega + \epsilon S_3(a, b, a_3, b_3, \dots) = 0.$$
(3d)

Eliminating ω^2 from the Eqs.(3b)-(3d) with the help of Eq.(3a), we get

$$\omega^{2} = \omega_{0}^{2} + b\mu\omega/a + \epsilon C_{1}(a, b, a_{3}, b_{3}, ...) - E/a, \tag{4a}$$

$$-b^{2}\mu\omega/a - a\mu\omega - \epsilon bC_{1}(a, b, a_{3}, b_{3}, ...) + \epsilon S_{1}(a, b, a_{3}, b_{3}, ...) + bE/a = 0,$$
(4b)

$$-8\omega_0^2 a_3 + 3b_3\mu\omega - 9a_3b\mu\omega/a - 9\epsilon a_3C_1(a,b,a_3,b_3,\ldots) + \epsilon C_3(a,b,a_3,b_3,\ldots) + 9a_3E/a = 0,$$
(4c)

$$-8\omega_0^2 b_3 - 3a_3\mu\omega - 9b_3b\mu\omega/a - 9\epsilon b_3C_1(a, b, a_3, b_3, ...) + \epsilon S_3(a, b, a_3, b_3, ...) + 9b_3E/a = 0.$$
(4d)

Now, using Eq.(4b), eliminating ω from the Eqs.(4c)-(4d) and taking only the linear terms of a_3 , b_3 and neglecting the terms of insignificant effects, we obtain two linear equations for a_3 and b_3 . From these equations a_3 and b_3 are determined. After putting

 a_3 and b_3 into Eq.(4b), b is expressed as a power series of small parameter $\lambda(\mu, \omega, E)$ in the following form:

$$b = m_0 + m_1 \lambda + m_2 \lambda^2 + m_3 \lambda^3 + \dots, \tag{5}$$

where m_0 , m_1 , m_2 are the functions of a. Finally, after putting a_3 , b_3 and b into Eq.(4a) and then solving this equation, the values of a are determined. Consequently, the desired values of b, a_3 and b_3 are calculated.

3 Example

Consider a generalized nonlinear (cubic-quintic) damped forced oscillator [29-33] of the following form:

$$\ddot{x} + \mu \dot{x} + x + \epsilon (\alpha_3 x^3 + \alpha_5 x^5) = E \cos(\omega t), \tag{6}$$

where $\omega_0^2=1$. According to the truncated Fourier series, the solution of Eq.(6) is assumed as [33]

$$x = a\cos(\omega t) + b\sin(\omega t) + a_3\cos(3\omega t) + b_3\sin(3\omega t) + \dots$$
 (7)

Putting Eq.(7) with its derivatives into Eq.(6) and then equating the coefficients of equal harmonics on both sides, we obtain

$$16(a + b\mu\omega - a\omega^{2}) + 12\epsilon((a^{2} - b^{2})a_{3} + 2aa_{3}^{2} + a(a^{2} + b^{2} + 2bb_{3} + 2b_{3}^{2})$$

$$+ 5\epsilon(6(a^{2} - b^{2})a_{3}^{3} + 6aa_{3}^{4} + 12aa_{3}^{2}(a^{2} + b^{2} + bb_{3} + b_{3}^{2}) + a_{3}(5a^{4} - 6a^{2}b^{2} - 3b^{4})$$

$$+ 6(a^{2} - b^{2})b_{3}^{2}) + 2a((a^{2} + b^{2})^{2} + 2(3a^{2}b + b^{3})b_{3} + 6(a^{2} + b^{2})b_{3}^{2} + 6bb_{3}^{3}$$

$$+ 3b_{3}^{4})(\alpha_{5}) = 16E,$$
(8a)

$$-16(a\mu\omega + b(-1+\omega^{2})) + 12\epsilon(b(a^{2}+b^{2}) - 2aba_{3} + 2ba_{3}^{2} + (a^{2}-b^{2})b_{3} + 2bb_{3}^{2})\alpha_{3}$$

$$+5\epsilon(2b(a^{2}+b^{2})^{2} - 12aba_{3}^{3} + 6ba_{3}^{4} + (3a^{4} + 6a^{2}b^{2} - 5b^{4})b_{3} + 12b(a^{2} + b^{2})b_{3}^{2}$$

$$+6(a^{2}-b^{2})b_{3}^{3} + 6bb_{3}^{4} - 4aba_{3}(a^{2} + 3b^{2} + 3b_{3}^{2}) + 6a_{3}^{2}(2b(a^{2} + b^{2})$$

$$+(a^{2}-b^{2})b_{3} + 2bb_{3}^{2}))\alpha_{5} = 0,$$
(8b)

$$48\mu\omega b_3 + \epsilon (30a(a^2 - 3b^2)a_3^2\alpha_5 + 10a_3^5\alpha_5 + 10a(a^2 - 3b^2)b_3^2\alpha_5 + a(a^2 - 3b^2)(4\alpha_3 + 5(a^2 + b^2)\alpha_5) + 4a_3^3((3\alpha_3 + 5(3(a^2 + b^2) + b_3^2)\alpha_5)) + 2a_3(8 - 72\omega^2 + 6\epsilon(2(a^2 + b^2) + b_3^2)\alpha_3) + 5\epsilon(3(a^2 + b^2)^2 + (6a^2b - 2b^3)b_3 + 6(a^2 + b^2)b_3^4)\alpha_5) = 0,$$
(8c)

$$10\epsilon a_3^4 b_3 \alpha_5 + \epsilon (-30b(-3a^2 + b^2)b_3^2 \alpha_5 + 10b_3^5 \alpha_5 + 12b_3^3 (\alpha_3 + 5(a^2 + b^2)\alpha_5))$$

$$-b(-3a^2 + b^2)(4\alpha_3 + 5(a^2 + b^2)\alpha_5)) + 2b_3(8 - 72\omega^2 + 12\epsilon(a^2 + b^2)\alpha_3$$

$$+15\epsilon(a^2 + b^2)^2 \alpha_5) - 4a_3(12\mu\omega - 5\epsilon a(a^2 - 3b^2)b_3\alpha_5) + 2\epsilon a_3^2(-5b(-3a^2 + b^2)\alpha_5)$$

$$+10b_3^3 \alpha_5 + 6b_3(\alpha_3 + 5(a^2 + b^2)\alpha_5)) = 0.$$
(8d)

Eliminating ω^2 from the Eqs.(8b)-(8d) with the help of Eq.(8a), and ignoring the terms whose responses are negligible, we obtain the following equations:

$$-16(-bE + a^{2}\mu\omega + b^{2}\mu\omega) - 3\epsilon(ba_{3}(3a^{2} - b^{2})(4\alpha_{3} + 5(a^{2} + b^{2})\alpha_{5}) -ab_{3}(a^{2} - 3b^{2})(4\alpha_{3} + 5(a^{2} + b^{2})\alpha_{5})) = 0,$$
(9a)

$$-4a_3(4(8a - 9E + 9b\mu\omega) + \epsilon a(21(a^2 + b^2)\alpha_3 + 15(a^2 + b^2)^2\alpha_5)) + a(48\mu\omega b_3 + \epsilon a(a^2 - 3b^2)(4\alpha_3 + 5(a^2 + b^2)\alpha_5)) = 0.$$
(9b)

$$-48a\mu\omega a_3 + \epsilon ab(3a^2 + b^2)(4\alpha_3 + 5(a^2 + b^2)\alpha_5) - 4b_3(4(8a - 9E + 9b\mu\omega) + \epsilon a((21(a^2 + b^2)\alpha_3 + 15(a^2 + b^2)\alpha_5)) = 0.$$
(9c)

Now, using Eq.(9b), eliminating ω from the Eqs.(9c) and (9d) and taking only the linear terms of a_3 , b_3 and omitting the terms whose response is negligible, we obtain

$$\epsilon a(a^4 - 2a^2b^2 - 3b^4)(4\alpha_3 + 5(a^2 + b^2)\alpha_5) - 4a_3(4(8a^2 + 8b^2 - 9aE)
+ \epsilon(21(a^2 + b^2)^2\alpha_3 + 15(a^2 + b^2)^3\alpha_5)) = 0,$$
(10a)

$$-\epsilon b(3a^4 + 2a^2b^2 - b^4)(4\alpha_3 + 5(a^2 + b^2)\alpha_5) + 4b_3(4(8a^2 + 8b^2 - 9aE) + \epsilon(21(a^2 + b^2)^2\alpha_3 + 15(a^2 + b^2)^3\alpha_5)) = 0.$$
(10b)

After solving Eqs. (10a) and (10b), a_3 and b_3 are determined as follows:

$$a_3 = \epsilon a(a^4 - 2a^2b^2 - 3b^4)(4\alpha_3 + 5(a^2 + b^2)\alpha_5)$$

$$/4(4(8a^2 + 8b^2 - 9bE) + \epsilon(21(a^2 + b^2)^2)\alpha_3 + 15(a^2 + b^2)^3\alpha_5),$$
(11a)

$$b_3 = \epsilon b(3a^4 + 2a^2b^2 - b^4)(4\alpha_3 + 5(a^2 + b^2)\alpha_5) /4(4(8a^2 + 8b^2 - 9bE) + \epsilon(21(a^2 + b^2)^2)\alpha_3 + 15(a^2 + b^2)^3\alpha_5).$$
(11b)

Inserting the values of a_3 and b_3 into Eq.(9a), we then expand b in a power series of the small parameter λ as follows:

$$b = l_0 + l_1 \lambda + l_2 \lambda^2 + l_3 \lambda^3 + \dots, \tag{12}$$

where $\lambda = 2\mu\omega/E$, $l_0 = a^2\mu\omega/E$, $l_1 = a^4\mu^2\omega^2/E^2$, $l_2 = 2a^6\mu^3\omega^3/E^3$, Finally,upon inserting a_3 , b_3 and b into Eq.(8a) and solving, the values of a are obtained. Consequently, the values b, a_3 and b_3 are determined.

4 Results and Discussion

The solutions determined by the present technique are compared with the corresponding numerical solution to justify the validity and the accuracy of the proposed technique. Comparisons between the solution curves obtained by the proposed method and a numerical method are shown graphically in Figures 1-4 in the presence of various damping and different values of the system parameters for strongly generalized nonlinear forced vibration problems.

From the figures, it is seen that the approximate results agree nicely with those solutions obtained by the numerical procedure.

5 Conclusion

In this study, a modified harmonic balance method is presented for handling strongly generalized nonlinear damped forced vibration problems. Some limitations of the classical HBM are overcome by the proposed method. The advantage of the present technique is that only one nonlinear algebraic equation is needed for solution. As a result, the computational effort is reduced and less effort is required than in the existing classical

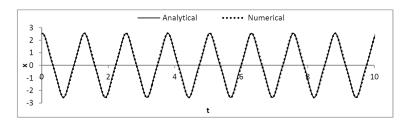


Figure 1: Comparison between the results obtained by the presented method and a numerical technique when $\omega = 5$, $\epsilon = 1.0$, $\alpha_3 = 1$, $\alpha_5 = 1$, $\mu = 0.1$, E = 10.

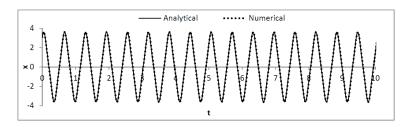


Figure 2: Comparison between the results obtained by the presented method and a numerical technique when $\omega = 10, \epsilon = 1.0, \alpha_3 = 1, \alpha_5 = 1, \mu = 0.25, E = 20.$

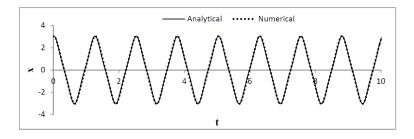


Figure 3: Comparison between the results obtained by the presented method and a numerical technique when $\omega = 5, \epsilon = 0.5, \alpha_3 = 1, \alpha_5 = 1, \mu = 0.05, E = 10.$

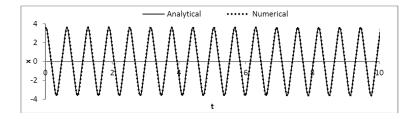


Figure 4: Comparison between the results obtained by the presented method and a numerical technique when $\omega=10, \epsilon=1.0, \alpha_3=1, \alpha_5=1, \mu=0.1, E=20.$

harmonic balance method. The results obtained by the present method show a good agreement with the numerical results. It is assumed that the proposed method is very effective and convenient for damped forced generalized nonlinear oscillators with strong as well as weak nonlinearities. Our results exhibit acceptable complaince with the solutions computed by the fourth order Runge-Kutta method for several values of systems parameters and significant damping.

Acknowledgment

The authors are grateful to the unknown reviewers for their valuable comments in preparing this research document.

References

- J. A. Wickert. Non-linear vibration of a traveling tensioned beam. *International Journal of Non-Linear Mechanics* 27 (1992) 503–517.
- [2] A. H. Nayfeh. Introduction to Perturbation Techniques. Wiley, New York, 1981.
- [3] M. Fooladi, S. R. Abaspour, A. Kimiaeifar and M. Rahimpour. On the analytical solution of Kirchhoff simplified Model for beam by using of homotopy analysis method. *World Applied Sciences Journal* **6** (2009) 297–302.
- [4] S. J. Liao. The proposed homotopy analysis technique for the solution of nonlinear problems. Ph.D. Thesis, Shanghai Jiao Tong University, 1992.
- [5] Y. Wu and J. H. He. Homotopy perturbation method for nonlinear oscillators with coordinate dependent mass. Results in Physics 10 (2018) 270–271.
- [6] M. A. Uddin, M. A. Alom and M. W. Ullah. An analytical approximate technique for solving a certain type of fourth order strongly nonlinear oscillatory differential system with small damping. Far East Journal of Mathematical Sciences 67 (1) (2012) 59–72.
- [7] C. U. Ghosh and M. A. Uddin. Analytical technique for damped nonlinear oscillators having generalized rational power restoring force. Far East Journal of Mathematical Sciences 130 (1) (2021) 25–41.
- [8] M. A. Wazwaz. The variational iteration method: A reliable analytic tool for solving linear and nonlinear wave equations. Computers and Mathematics with Applications 54 (2007) 926–932.
- [9] J. H. He. Some asymptotic methods for strongly nonlinear equations. *International Journal Modern Physics B* 20 (2006) 1141–1199.
- [10] R. E. Mickens. A generalization of the method of harmonic balance. Journal of Sound and Vibration 111 (1986) 515–518.
- [11] K. Y. Sze, S. H. Chen and J. L. Huang. The incremental harmonic balance method for nonlinear vibration of axially moving beams. *Journal of Sound and Vibration* 281 (2005) 611–626.
- [12] A. S. M. Z. Hasan, M. S. Rahman, Y. Y. Lee and A. Y. T. Leung. Multi-level residue harmonic balance solution for the nonlinear natural frequency of axially loaded beams with an internal hinge. *Mechanics of Advanced Materials and Structures* 24 (13) (2017) 1074– 1085
- [13] Y. Y. Lee. Analytic solution for nonlinear multimode beam vibration using a modified harmonic balance approach and vietas substitution. *Shock and Vibration* (2016) Article ID: 3462643.

- [14] A. H. Nayfeh. Perturbation Method. John Wiley and Sons, New York, 1973.
- [15] A. H. Nayfeh and D.T. Mook. Nonlinear oscillations. John Wiley and Sons, New York, 1979.
- [16] N. N. Krylov and N. N. Bogoliubov. Introduction to Nonlinear Mechanics. Princeton University Press, New Jersey, 1947.
- [17] M. S. Alam. A unified Krylov-Bogoliubov-Mitropolskii method for solving nth order non-linear systems. *Journal of the Franklin Institute* 399 (2) (2002) 239–248.
- [18] M. S. Alam, M. A. K. Azad and M. A. Hoque. A general Struble's technique for solving an nth order weakly nonlinear differential system with damping. *International Journal of Nonlinear Mechanics* 41 (2006) 905–918.
- [19] M. A. Razzak and M. H. U. Molla. A Simple Analytical Technique to Investigate Nonlinear Oscillations of an Elastic Two Degrees of Freedom Pendulum. *Nonlinear Dynamics and Systems Theory* 15 (4) (2015) 409–417.
- [20] S. E. Jones. Remarks on the perturbation process for certain conservative systems. International Journal of Non-Linear Mechanics 13 (1978) 125–128.
- [21] Y. K. Cheung, S. H. Chen and S. L. Lau. A modified Lindstedt-Poincare method for certain strongly nonlinear oscillators *International Journal of Non-Linear Mechanics* 26 (1991) 367–78.
- [22] M. S. Alam, I. A. Yeasmin and M. S. Ahamed. Generalization of the modified Lindstedt-Poincare method for solving some strongly nonlinear oscillators. Ain Shams Engineering Journal 10 (1) (2019) 195–201.
- [23] M. S. Rahman, M. E. Haque and S. S. Shanta. Harmonic balance solution of nonlinear differential equation (non-conservative). *Journal of Advances in Vibration Engineering* 9 (4) (2010) 345–356.
- [24] S. L. Lau and Y. K. Cheung. Amplitude incremental variational principle for nonlinear structural vibrations. *Journal of Applied Mechanics* 48 (1981) 959–964.
- [25] L. Azrar, R. Benamar and R. G. White. A semi-analytical approach to the nonlinear dynamic response problem of beams at large vibration amplitudes, Part II: multimode approach to the steady state forced periodic response. *Journal of Sound and Vibration* 255 (1) (2002) 1–4.
- [26] M. S. Rahman and Y. Y. Lee. New modified multi-level residue harmonic balance method for solving nonlinearly vibrating double-beam problem. *Journal of Sound and Vibration* 6 (2017) 295–327.
- [27] Y. Wu. The harmonic balance method for Yao-Cheng oscillator. Journal of Low Frequency Noise Vibration and Active Control 38 (2019) 1716–1718.
- [28] U. V. Wagner and L. Lentz. On the detection of artifacts in harmonic balance solutions of nonlinear oscillators. Applied Mathematical Modeling 65 (2019) 408–414.
- [29] D. Younesian, H. Askari, Z. Saadatnia and M. K. Yazdi. Frequency analysis of strongly nonlinear generalized Duffing oscillators using He's frequency amplitude formulation and He's energy balance method. Computers and Mathematics with Applications 59 (2010) 3222–3228.
- [30] M. A. Uddin, M. W. Ullah and R. S. Bipasha. An approximate analytical technique for solving second order strongly nonlinear generalized Duffing equation with small damping. *Journal of Bangladesh Academy of Sciences* 39 (1) (2015) 103–114.
- [31] H. Rafieipour, A. Lotfavar and A. Masroori. Analytical approximate solution for nonlinear vibration of microelectro mechanical system using He's frequency amplitude formulation. *IJST, Transactions of Mechanical Engineering* **37(M2)** (2013) 83–90.

- [32] M. M. F. Karahan and M. Pakdemirli. Free and forced vibrations of the strongly nonlinear cubic-quintic Duffing oscillators. *Zeitschriftfr Naturforschung A* **72**(1) (2017) 59-69.
- [33] M. W. Ullah, M. S. Rahman and M. A. Uddin. A modified harmonic balance method for solving forced vibration problems with strong nonlinearity. *Journal of Low Frequency Noise*, *Vibration and Active Control* **40(1)** (2021) DOI: 10.1177/1461348420923433.
- [34] M. S. Rahman, A. S. M. Z. Hasan and I. A. Yeasmin. Modified multi-level residue harmonic balance method for solving nonlinear vibration problem of beam resting on nonlinear elastic foundation. *Journal of Applied and Computational Mechanics* 5 (4) (2019) 627–638.
- [35] I. A. Yeasmin, N. Sharif, M. S. Rahman and M. S. Alam. Analytical technique for solving the quadratic nonlinear oscillator. *Results in Physics* 18 (2020) Article No. 103303.
- [36] A. K. Cheib, V. E. Puzyrov and N. V. Savchenko. Analysis of the Dynamics of a Two-Degree-of-Freedom Nonlinear Mechanical System under Harmonic Excitation. *Nonlinear Dynamics and Systems Theory* **20** (2) (2020) 168–178.